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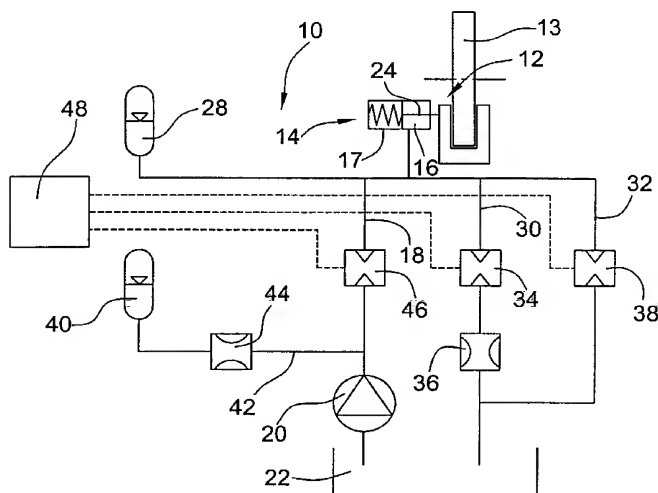
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(54) Title: HYDRAULIC BRAKE SYSTEM FOR A WIND ENERGY PLANT



(57) Abstract: The hydraulic brake system comprises a piston/cylinder unit (14) for activating a brake (12), comprising a volume (16) adapted to have a pressurized working fluid supplied thereto for moving a piston (24) from a braking position into a release position for releasing the brake. The hydraulic brake system comprises a supply line (18) for the supply of pressurized working fluid, a further supply line (42) connected to the volume (16) of the piston/cylinder unit (14), a storage means (40) for pressurized working fluid, connected to the further supply line (42), an ON/OFF valve (46) arranged in the further supply line (42). At the start of a braking process the ON/OFF valve (46), which in the released condition of the brake is in the opened condition, is closed and at the end of a braking process, when the rotational speed of the component (13) to be braked has decreased to a presettable value, a discharge of working fluid from the volume (16) is prevented, and the further ON/OFF valve (46) for supply of pressurized working fluid from the storage means (40) into the volume (16) is opened.



WO 03/080414 A1

TITLE OF THE INVENTION

Hydraulic brake system for a wind energy plant

FIELD OF THE INVENTION

The invention relates to a hydraulic brake system for braking a moving component of a wind energy plant, such as the brake disk of the power train or the azimuth drive of the wind energy plant.

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BACKGROUND OF THE INVENTION

In several technical applications hydraulic brake systems are used, wherein, in particular in cases of emergency, a rotating component must for reasons of safety be braked down to a standstill within a very short period of time. An
10 example of such a technical application is a wind energy plant whose rotor must be very rapidly braked in the case of malfunctions.

In order to meet the requirements of rapid braking, a relatively large braking force should be applied, which may however lead to considerable vibrations
15 and excessive dynamic load peaks in the system to be braked. This, in turn, may cause damage to the system and lead to relatively large material fatigue and aging. Vibrations and excessive dynamic load peaks also occur at the end of a braking process (stoppage vibrations).

20 From WO-A-98/23474 a hydraulic brake system for, *inter alia*, wind energy plants is known, which ensures smooth braking of the rotor via a relatively complex valve drive.

SUMMARY OF THE INVENTION

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It is an object of the present invention to provide a hydraulic brake system for braking a rotating component of a wind energy plant, the brake system, which

is of a very simple configuration, allowing for relatively low-vibration and in any case reliable braking of the component at the end of a braking process.

To achieve this object the invention suggests, according to a first aspect, a hydraulic brake system comprising:

- at least one piston/cylinder unit for activating and releasing a brake,
- the piston being biased into a braking position for activating the brake and the cylinder comprising a volume adapted to have a pressurized working fluid supplied thereto for moving the piston from the braking position into a release position for releasing the brake,
- a supply line connected to the volume of the at least one piston/cylinder unit for the supply of working fluid from a reservoir,
- a pump arranged in the supply line,
- a further supply line connected with the volume of the at least one piston/cylinder unit,
- a storage means for pressurized working fluid, connected to the further supply line,
- an ON/OFF valve arranged in the further supply line, and
- a drive unit being adapted to cause working fluid to be discharged from the volume of the at least one piston/cylinder unit at the start of a braking process,
- wherein the drive unit at the start of a braking process closes the ON/OFF valve which in the released condition of the brake is in the opened condition, and
- wherein the drive unit at the end of a braking process, if the rotational speed of the component to be braked has decreased to a presettable value, prevents a discharge of working fluid from the volume of the at least one piston/cylinder unit and opens the further ON/OFF valve for supply pressurized working fluid from the storage means to the volume of the at least one piston/cylinder unit.

Alternatively, the brake system comprises:

- at least one piston/cylinder unit for activating and releasing a brake,
- the piston being biased into a braking position for activating the brake and the cylinder comprising a volume adapted to have a pressurized working fluid supplied thereto for moving the piston from the braking position into a release position for releasing the brake,
- a supply line connected to the volume of the at least one piston/cylinder unit for the supply of working fluid from a reservoir,
- a pump arranged in the supply line,
- a further supply line connected to the volume of the at least one piston/cylinder unit,
- a storage means for pressurized working fluid, connected to the further supply line,
- an ON/OFF valve arranged in the further supply line,
- a discharge line for the discharge of working fluid from the volume of the at least one piston/cylinder unit,
- an ON/OFF valve arranged in the discharge line, and
- a drive unit for the two ON/OFF valves,
- wherein the drive unit is adapted to cause working fluid to be discharged from the volume of the at least one piston/cylinder unit at the start of the braking process,
- wherein the drive unit at the start of a braking process closes the ON/OFF valve in the further supply line, which in the released condition of the brake is in the opened condition, and, at the start of a braking process opens the ON/OFF valve in the discharge line, which in the released condition of the brake is in the closed condition, and
- wherein the drive unit at the end of a braking process, if the rotational speed of the component to be braked has decreased to a presettable value, closes the ON/OFF valve in the discharge line to prevent a discharge of working fluid from the volume of the at least one piston/cylinder unit, and opens the further ON/OFF valve for supply of pressurized working fluid from the storage means into volume of the at least one piston/cylinder unit.

This first embodiment comprises, according to its two alternatives, a passive brake which automatically acts upon the component to be braked when the working fluid pressure decreases. For this purpose, the piston of the at least one piston/cylinder unit is e. g. mechanically biased by a spring into a release position. The volume of the piston/cylinder unit is normally supplied with pressurized working fluid which is fed by a pump via a supply line and kept under pressure thus maintaining the brake in the open position against the spring bias.

To prevent the brake from automatically falling into the operative position in the case of pump failure, a storage means is connected via a line with the volume of the piston/cylinder unit, said storage means being supplied by the pump with pressurized working fluid. In the case of pump failure, the pressure of the working fluid in the volume of the piston/cylinder unit is maintained via the storage means. This aspect is essential for reasons of safety, it is however of minor importance to the invention.

To allow activation of the passive brake in a case of emergency, working fluid can escape via a discharge line from the volume of the piston/cylinder unit. To prevent this process from occurring all too suddenly, the discharge line is on the one hand adapted to be opened and closed, for which purpose an ON/OFF valve is arranged in the discharge line. On the other hand, the flow resistance of the working fluid in the discharge line is increased by arranging an element in the discharge line or by the configuration of the valve, such that the working fluid cannot all too suddenly flow out of the volume of the piston/cylinder unit. The element determining the flow resistance is in particular a constriction, i. e. a reduction of the cross-section.

Concerning the operating safety, it is necessary that enough working fluid can be discharged via the discharge line per unit of time to allow this brake to fall into the operative position in due time and to a large enough extent. To

ensure this even in the case of e. g. an unintended reduction of the flow cross-section due to deposits in the valve and/or the element and/or the line, the invention provides a further valve which is arranged in a bypass line connected in parallel to the discharge line, and is closed at the start of a
5 braking process, said valve being opened in a time-controlled manner after opening of the valve in the discharge line. The time period is presettable. This forced opening of the second valve ensures that actually as much working fluid as is required for final activation of the brake can escape from the volume of the piston/cylinder unit, irrespective of whether working fluid has
10 already escaped from the volume of the piston/cylinder unit, or of the amount of escaped working fluid.

Vibrations do not only occur at the start of a braking process but also at the end of said process (so-called stoppage vibrations). Thus, according to the
15 invention, the brake is released before the component to be braked comes to a final standstill.

The storage means provided by the invention differs from the storage means described above and can in particular be realized by the pump and the
20 reservoir. The further supply line from the storage means to the volume of the piston/cylinder unit is normally open.

At the start of a braking process the (further) ON/OFF valve is closed and the other valves are driven as described above via the drive unit. When the
25 presettable minimum rotational number or minimum speed of the component to be braked is reached, the valves in the discharge and the bypass line are closed and the further valve is opened such that working fluid flows from the storage means into the volume of the piston/cylinder unit and releases again the passive brake. This counteracts the generation of stoppage vibrations.

30

To prevent the (re-) filling of the volume of the piston/cylinder unit from occurring all too suddenly, it is appropriate to arrange an element in the

discharge line for increasing the flow resistance, which is realizable by a corresponding configuration of the further valve, the further supply line, or arrangement of a component, e. g. a constriction.

- 5 Preferably, the brake is activated again after its release, which is appropriately performed in the same manner as at the start of a braking process. Thus the drive unit closes the further valve and opens the valve in the discharge line. Preferably, this process is followed by a time-controlled opening of the valve in the bypass line.

10

Instead of a passive brake, the brake system according to the invention may comprise an active brake whose at least one piston/cylinder unit must be supplied with pressurized working fluid for activation purposes. This embodiment of the invention is provided with:

- 15 - at least one piston/cylinder unit for activation and release of a brake,
- the cylinder comprising a volume for a working fluid, which is provided to have a pressurized working fluid supplied thereto,
- a supply line connected to the volume of the at least one piston/cylinder unit for the supply of pressurized working fluid from a reservoir,
20 - a pump arranged in the supply line,
- a discharge line provided with a further ON/OFF valve and arranged in flow connection with the volume of the at least one piston/cylinder unit,
- a drive unit for triggering the inflow of working fluid into the volume of the at least one piston/cylinder unit at the start of a braking process,
25 - wherein the ON/OFF valve, closed in the released condition of the brake, remains in its closed condition at the start of a braking process, and wherein the drive unit opens the ON/OFF valve at the end of a braking process.

30 Alternatively, the brake system comprises:

- at least one piston/cylinder unit for activating and releasing a brake,

- the cylinder comprising a volume for a working fluid adapted to have a pressurized working fluid supplied thereto,
 - a supply line connected to the volume of the at least one piston/cylinder unit for the supply of working fluid from a reservoir,
 - 5 - an ON/OFF valve arranged in the supply line,
 - a pump arranged in the supply line,
 - a discharge line provided with a further ON/OFF valve and arranged in flow connection with the volume of the at least one piston/cylinder unit,
 - a drive unit for the two ON/OFF valves,
 - 10 - wherein the drive unit for the inflow of working fluid into the volume of the at least one piston/cylinder unit opens the ON/OFF valve in the supply line at the start of a braking process,
 - wherein the ON/OFF valve in the discharge line, closed in the released condition of the brake, remains in its closed condition at the start of a
 - 15 braking process, and
 - wherein at the end of a braking process, the drive unit, if the rotational speed of the component to be braked has decreased to a presettable value, opens the ON/OFF valve in the discharge line.
- 20 In these embodiments, an ON/OFF valve and an element determining the flow resistance are provided in the supply line connecting the pump with the volume of the piston/cylinder unit. A bypass line provided with an ON/OFF valve is arranged in parallel to this portion of the supply line. At the start of the braking process, first the valve in the supply line opens such that, as
- 25 determined by the flow resistance, working fluid flows into the volume of the piston/cylinder unit at a specified rate for "smooth" actuation of the active brake. In a time-controlled manner the valve in the bypass line opens upon lapse of a presettable time period. Thus a reliable falling into the operative position of the active brake is guaranteed.
- 30 For reasons of operating safety, the pump pumps working fluid into a storage means branching off the supply line behind the pump and in front of the valve

or the element determining the flow resistance. In the case of pump failure, the storage means contains a sufficient amount of working fluid for filling all piston/cylinder units of the brake. This aspect of a hydraulic brake system comprising an active brake is of minor importance to the invention.

5

The embodiment of the invention described above helps to counteract vibrations and excessive dynamic load peaks generated in the system to be braked at the start of a braking process. This embodiment can be extended by a (additional) (discharge) line provided with a valve in order to reduce stoppage vibrations occurring at the end of a braking process.

10

Via the discharge line branching off e. g. the bypass line or the supply line working fluid can be discharged from the piston/cylinder unit by opening the further valve. This process releases the brake. For subsequent activation of the brake for the purpose of fixing the component to be braked when said component has stopped, the further valve is closed again such that working fluid can no longer flow out of the volume of the piston/cylinder unit, and thus this volume is refilled with working fluid from the storage means.

15

20

BRIEF DESCRIPTION OF THE DRAWINGS

Hereunder the invention is explained in detail with reference to the drawings in which:

25

Fig. 1 shows the circuit diagram for a hydraulic brake system comprising a passive (fail-safe) brake,

Fig. 2 shows a schematic representation of the time sequence for driving the valves of the system shown in Fig. 1,

30

Fig. 3 shows the circuit diagram for a hydraulic brake system comprising an active brake, and

Fig. 4 shows a schematic representation of the time sequence for driving the valves of the system shown in Fig. 3.

DETAILED DESCRIPTION OF PREFERRED EMBODIMENTS

10 Figs. 1 and 2 relate to a first embodiment of a hydraulic brake system 10. System 10 comprises a passive brake 12 for a - in this case, rotating - component 13 (e.g. brake disk on the fast or slow rotor shaft of a wind energy plant) with one or a plurality of piston/cylinder units 14. The working volumes 16 of the cylinders 17 of the piston/cylinder unit 14 are supplied with
15 pressurized working fluid from a reservoir 22 via a supply line 18 by means of a pump 20. The piston 24 of each piston/cylinder unit 14 is biased, e.g. by means of a spring 26, into the braking position actuating the brake 12, and can be moved against the biasing force into a release position by pumping working fluid into the working volume. Connected to the working volume 16 is
20 a first storage means 28 storing such a quantity of pressurized working fluid that the overall volume of the brake piston and the storage means will always be sufficient to close the brake 12.

Connected to the working volume 16 of each piston/cylinder unit 14 are a
25 discharge line 30 and a bypass line 32 connected in parallel thereto, which lines lead back to the reservoir 22. The discharge line 30 has an ON/OFF valve 34 and a constriction 36 arranged therein, and the bypass line 32 has an ON/OFF valve 38 arranged therein.

30 Arranged to enter the supply line 18 is a further (supply) line 42 connected to a further storage means 40 and provided with a further constriction 44. This line 42 is connected to the supply line 18 between the pump 20 and a further

ON/OFF valve 46 arranged behind the pump 20 in the flow direction of the working fluid. To carry out a braking process, all valves 34, 38 and 46 are controlled by a drive unit 48 in the sequence represented in Fig. 2.

- 5 Prior to a braking process, valves 34 and 38 are closed and valve 46 is open. Pump 20 supplies pressurized working fluid to the two storage means 28 and 40 and to each piston/cylinder unit 14 of the passive brake 12.

10 At the beginning T_1 of a brake process, valve 34 is opened so that working fluid can flow off from each piston/cylinder unit 14. The flow-off rate is determined by the constriction 36 and is selected such that the brake 12 does not act too promptly onto the component 13 to be braked. In this manner, excessive vibrations and excessive load peaks of the system to be braked (not shown) are avoided.

15

Displaced in time relative to the beginning T_1 of the brake process by the delay period T_{delay1} , the valve 38 of the bypass line 32 positively opened by the drive unit 48 at the point of time T_2 ($T_2 = T_1 + T_{\text{delay1}}$). Thereby, it is safeguarded that at the latest after the lapse of the time period T_{delay1} ,
20 working fluid possibly still contained in the piston/cylinder units 14 which prevents a complete effectiveness of the brake, can flow off. this can be the case e.g. when the discharge line is at least partially clogged.

The delay period T_{delay1} is selected such that, up to its lapse, working fluid will
25 already have flown through the opened valve 34 and the constriction 36 out of the volume 16 in such an extent that the brake 12 is activated. Thus, the main purpose of valve 38 resides in a safety activation of brake 12 in case that, due to a (partial) clogging of the constriction 36 within period T_{delay1} , the prescribed quantity of working fluid has not yet flown off from the volume 16,
30 so that this (residual) working fluid can flow off via valve 38 and the brake 12 will in any case be active after lapse of the period T_{delay1} .

Towards the end of the braking process, valve 46 is opened at a time T_3 while valves 34 and 38 are closed. Now, working fluid is moved out of the storage means 40 with a flow rate determined by constriction 44 into the volumes 16 of the piston/cylinder unit 14 of the passive brake 12, whereby the latter is released again. At the time T_3 , the component 13 to be braked still has a residual rotational speed, i.e. is not yet at a standstill. By the release of brake 12 at this point of time, a generation of stoppage vibrations is prevented.

The size of storage means 40 has to be selected such that a sufficient quantity of pressurized working fluid will be available so that the working volumes 16 of piston/cylinder unit 14 can be filled for releasing the brake 12 and, in the given case, the storage means 28 can be "overpressed".

After the brake has thus been released, it should suitably be actuated again for final arrest of component 13 (locking brake function). For this purpose, valve 46 is closed and valve 34 is opened at a point of time T_4 provided by a period of time $T_{\text{delay}2}$ after point of time T_3 . Now, working fluid flows off again from the volumes 16 of piston/cylinder unit 14 with a flow-off rate determined by the constriction 36. Upon lapse of a time period $T_{\text{delay}3}$ subsequent to the point of time T_4 , valve 38 is opened at the point of time T_5 ($T_5 = T_4 + T_{\text{delay}3}$) whereby the final activation of brake 12 for fixing the component 13 is guaranteed even if, during the time period $T_{\text{delay}3}$, not all of the working fluid should have been flown off yet from the working volumes 16 of piston/cylinder unit 14 via discharge line 30.

The individual time periods $T_{\text{delay}1}$, $T_{\text{delay}2}$ and $T_{\text{delay}3}$ can be identical or different from each other and should be selected corresponding to the system conditions and under performance aspects.

Figs. 3 and 4 illustrate an embodiment of a hydraulic brake system 60 with an active brake 62 for braking a component 63 arranged e.g. as a brake disk of a rotor shaft of a wind energy plant. Brake 62 comprises at least one the at

least one piston/cylinder unit 64 with a cylinder 66, a working volume 68 and a piston 70. Connected to the working volume 68 is a supply line 72 having a pump 74 for a working fluid arranged therein. Pump 74 is arranged for the pressurized pumping of working fluid from a reservoir 76 into the working
5 volume 68 of the piston/cylinder unit(s) 64 when the brake is to be actuated. Connected to the supply line 72 is a storage means 78 which is supplied with pressurized working fluid by pump 74. In case of malfunction of pump 74, it is still possible to actuate the brake 62 via the storage means 78.

10 In the supply line 72, an ON/OFF valve 80 and a constriction 82 are arranged. Thus, when the brake 62 is actuated, the supply line 72 can be opened and working fluid can flow into the working volume 68 of each piston/cylinder unit 64 at a rate determined by constriction 82.

15 Arranged in parallel to supply line 72 is a bypass line 84 connected to each piston/cylinder unit 64 and provided with an ON/OFF valve 86. At a site downstream of pump 74 with respect to the flow direction of the working fluid, this bypass line 84 is connected to supply line 72. The bypass line 84 has a discharge line 88 branching off it, with an ON/OFF valve 90 and a constriction
20 92 arranged therein.

Using the above described system 60, a braking profile is obtained wherein vibrations and excessive loads in the drive line of the system to be braked can be suppressed both at the beginning and at the end of the braking process;
25 the sequence of the valves 80, 86 and 90 driven by a drive unit 94 is shown in Fig. 4.

In the normal case, i.e. when the brake is not actuated, all of the above three valves 80, 86 and 90 are closed. The storage means 78 is provided with
30 pressurized working fluid.

At the beginning T_1 of a brake process, valve 80 is opened so that working fluid will be introduced into the working volume 68 from the storage means 78 in case of a failure of pump 74 and from the reservoir 76 in case of an intact condition of pump 74; the flow rate is determined by the constriction 82 in a manner avoiding an all to sudden actuation of brake 62. After a
5 predetermined time period T_{delay} from the time T_1 valve 86 is positively opened (time T_2), thus safeguarding that the volume 68 of each piston/cylinder unit 64 will now in any case be supplied with working fluid to effect the maximum possible actuation of brake 62, in as far as this has not been carried out
10 before via supply line 72 due to disturbances.

Setting the flow rate in supply line 72 through the constriction 82 allows for a "soft" initiating of the brake process, precluding the generation of vibrations and excessive load peaks in the braked system. The bypass line 84 opened
15 after a time delay is relevant under safety aspects and should in the normal case have no influence anymore on the actuation of the brake.

To avoid or reduce stoppage vibrations, it is provided that, at a time T_3 when the rotational speed of the to-be-braked component 63 has already decreased
20 to a predetermined value, the valve 90 of the discharge line is opened so that working fluid can escape from the piston/cylinder unit 68 via the supply line 72 and/or the bypass line 84 (with both or at least one of the valves 80 and 86 being open) into the discharge line 88 and from there into the reservoir 76. Thus, the brake 62 is released. This process can be terminated after a short
25 span of time so that the brake 62 will again resume its full frictional engagement with the to-be-braked component 63 to arrest the same (in a condition of standstill). The refilling of the piston/cylinder units 64 can be performed first via the supply line 72 (with valve 80 opened and valve 86 still closed) and thereafter also via the bypass line (with valve 86 opened at this
30 time). This allows for a smooth and in any case safeguarded (renewed) actuation of brake 62 so that the brake will then act as a locking brake.

CLAIMS

1. A hydraulic brake system for braking a moving component in a wind energy plant, comprising
 - at least one piston/cylinder unit (14) for activating a brake (12),
 - the piston (24) being biased into a braking position for activating the brake (12) and the cylinder (17) comprising a volume (16) adapted to have a pressurized working fluid supplied thereto for moving the piston (24) from the braking position into a release position for releasing the brake,
 - a supply line (18) connected to the volume (16) of the at least one piston/cylinder unit (14) for the supply of pressurized working fluid from a reservoir (22),
 - a pump (20) arranged in the supply line (18),
 - a second supply line (42) connected to the volume (16) of the at least one piston/cylinder unit (14),
 - a storage means (40) for pressurized working fluid, connected to the further supply line (42),
 - an ON/OFF valve (46) arranged in the further supply line (42),
 - a discharge line (30;32) for the discharge of working fluid from the volume (16) of the at least one piston/cylinder unit (14),
 - an ON/OFF valve (34;38) arranged in the discharge line (30;32), and
 - a drive unit (48) for the two ON/OFF valves (36,30;46,32),
 - the drive unit (48) being adapted to cause working fluid to be discharged from the volume (16) of the at least one piston/cylinder unit (14) at the start of a braking process,
 - the drive unit (48) at the start of a braking process closes the ON/OFF valve (46) in the further supply line (42), which in the released condition of the brake (12) is in the opened condition, and at the start of a braking process opens the ON/OFF valve (34;38), which in the released condition of the brake is in the closed condition, and

- the drive unit (48) at the end of a braking process, if the rotational speed of the component (13) to be braked has decreased to a presettable value, closes the ON/OFF valve (30;32) in the discharge line (30;32) to prevent a discharge of working fluid from the volume (16) of the at least one piston/cylinder unit (14), and opens the further ON/OFF valve (46) for supply of pressurized working fluid from the storage means (40) into the volume (16) of the at least one piston/cylinder unit (14).
2. The hydraulic brake system according to claim 1, wherein the supply line (18) is arranged to enter the further supply line (42) between the storage means (40) and the further ON/OFF valve (46).
 3. The hydraulic brake system according to claim 1 or 2, wherein the drive unit (48) at the end of the braking process upon lapse of a presettable time period ($T_{\text{delay}2}$) from the opening of the ON/OFF valve (46) arranged in the further supply line (42), closes the same again and opens the ON/OFF valve (34;38) in the discharge line (30;32) for subsequent renewed discharge of working fluid from the volume (16) of the at least one piston/cylinder unit (14).
 4. The hydraulic brake system according to any one of claims 1 to 3, wherein the further supply line (42) has arranged therein an element (44) determining the flow resistance.
 5. A hydraulic brake system for braking a moving component in a wind energy plant, comprising
 - at least one piston/cylinder unit (64) for activating and releasing a brake (62),
 - the cylinder (66) comprising a volume (68) for a working fluid, provided to have a pressurized working fluid supplied thereto for activating the brake (62),

- a supply line (72;84) connected to the volume (68) of the at least one piston/cylinder unit (64) for the supply of pressurized working fluid from a reservoir (76),
 - an ON/OFF valve (80;86) arranged in the supply line (72),
 - a pump (76) arranged in the supply line (72),
 - a discharge line (88) provided with an ON/OFF valve (90) and arranged in flow connection with the volume (68) of the at least one piston/cylinder unit (64),
 - a drive unit (94) for the two ON/OFF valves (80,90;86,90),
 - at the start of a braking process, the drive unit (94) opens the ON/OFF valve (80;86) in the supply line (72) to cause working fluid to flow into the volume (68) of the at least one piston/cylinder unit (64),
 - the ON/OFF valve (90) in the discharge line (90), closed in the released condition of the brake, being provided to remain in its closed condition at the start of a braking process, and
 - the drive unit (94) at the end of a braking process, if the rotational speed of the component (63) to be braked has decreased to a presettable value, closes the ON/OFF valve (90) in the discharge line (88).
6. The hydraulic brake system according to claim 5, wherein the discharge line (88) has arranged therein an element (92) determining the flow resistance.
7. The hydraulic brake system according to claim 1 to 6, wherein each element (36,44,82,92) determining the flow resistance is provided as a cross-sectional constriction in an ON/OFF valve and/or in a line or as a flow deflector.

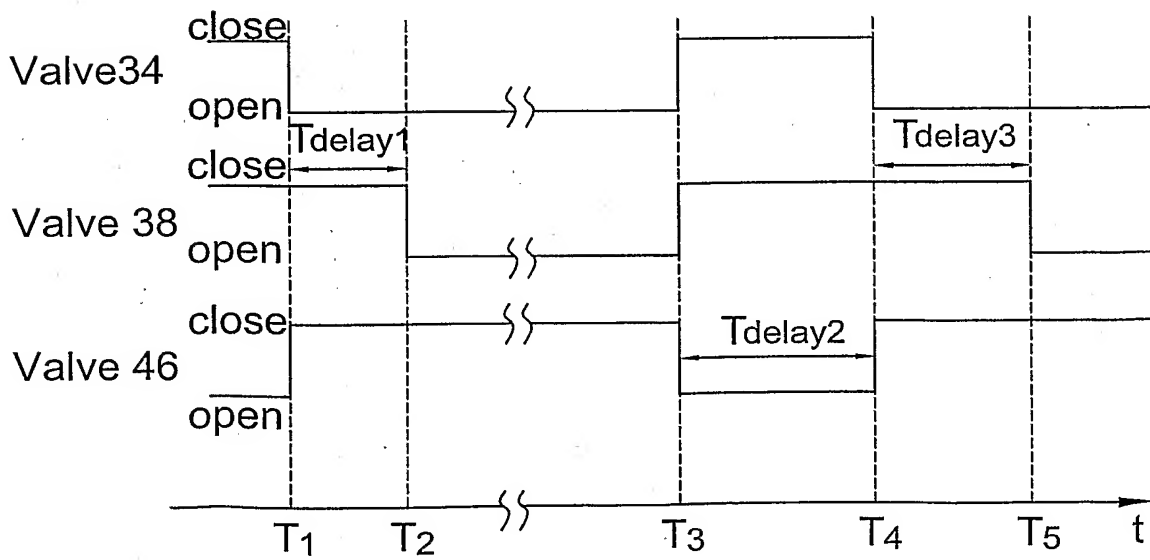
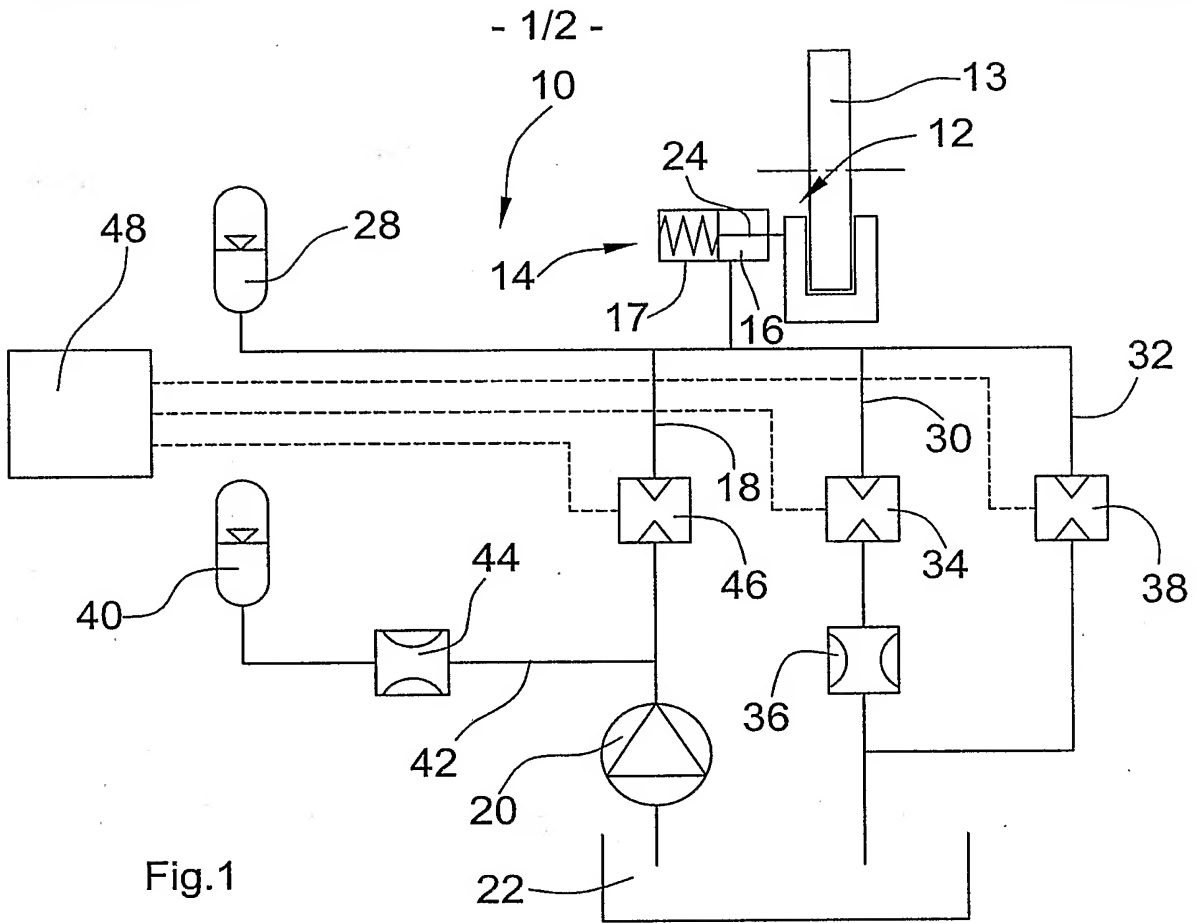


Fig.2

- 2/2 -

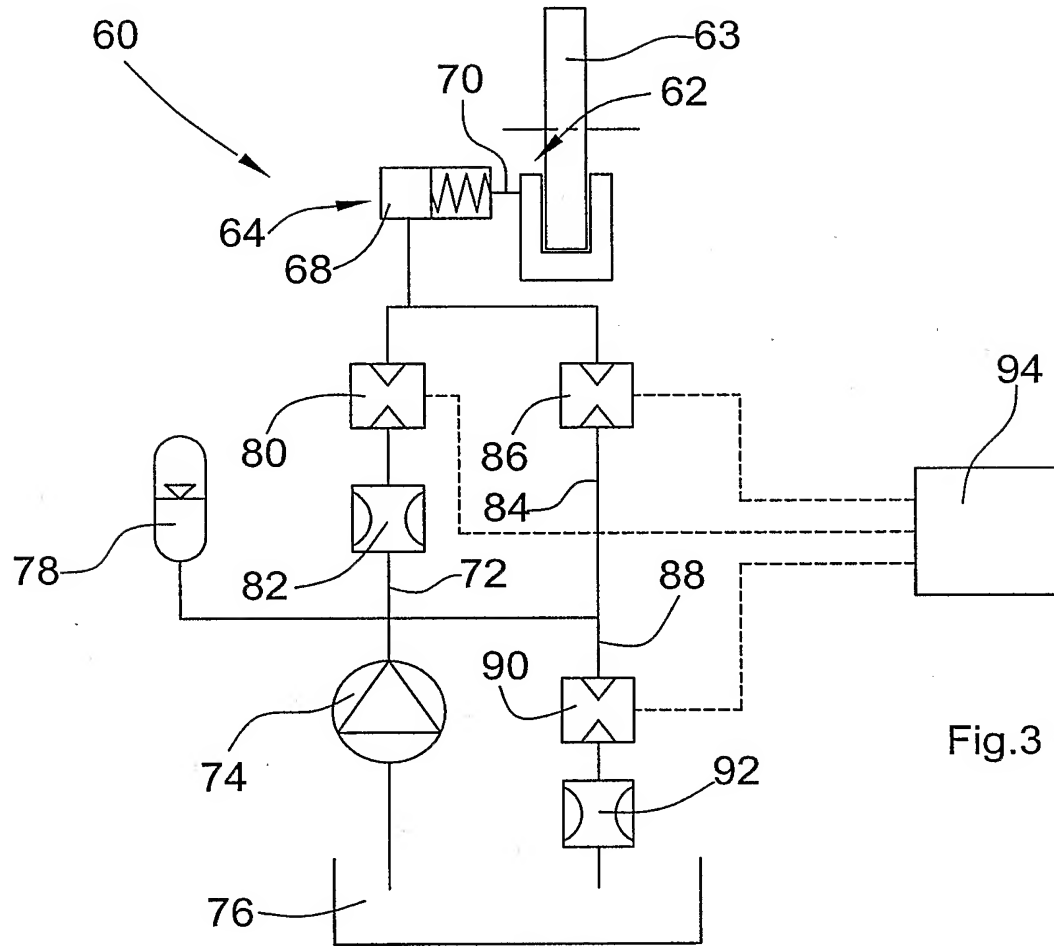


Fig.3

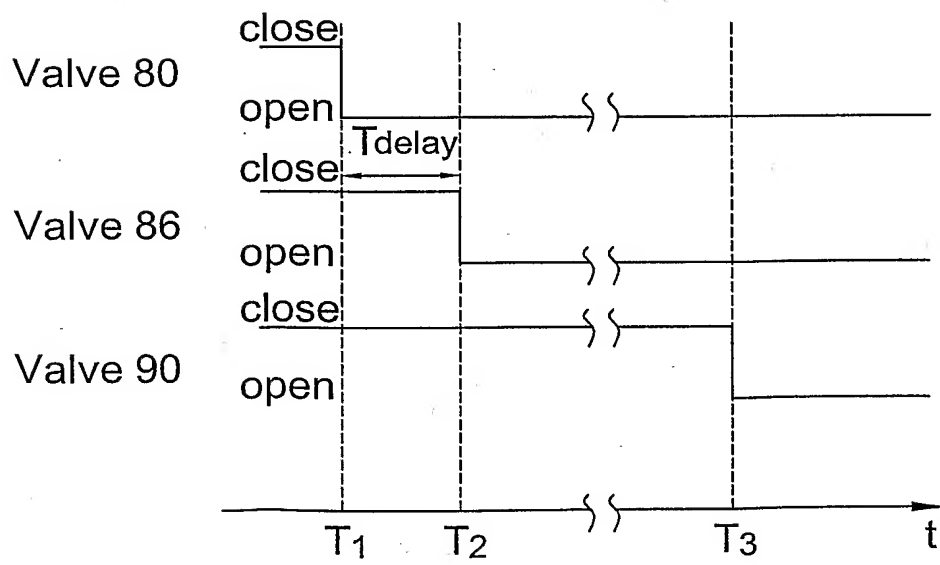


Fig.4

INTERNATIONAL SEARCH REPORT

International Application No

PCT/EP 03/03047

A. CLASSIFICATION OF SUBJECT MATTER
IPC 7 B60T13/22 B60T13/68 F03D7/02

According to International Patent Classification (IPC) or to both national classification and IPC

B. FIELDS SEARCHED

Minimum documentation searched (classification system followed by classification symbols)

IPC 7 B60T F03D

Documentation searched other than minimum documentation to the extent that such documents are included in the fields searched

Electronic data base consulted during the international search (name of data base and, where practical, search terms used)

EPO-Internal, WPI Data

C. DOCUMENTS CONSIDERED TO BE RELEVANT

Category *	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
X	US 6 254 197 B1 (SANGILL OLE ET AL) 3 July 2001 (2001-07-03) cited in the application column 2, line 16 - line 42 column 2, line 64 - column 3, line 15 column 3, line 48 - column 4, line 23 column 5, line 1 - line 6 column 6, line 67 - column 7, line 12 column 8, line 6 - line 53 column 8, line 59 - line 65 column 9, line 40 - line 59 column 9, line 11 - line 16 column 10, line 6 - line 25; figure 1 ---	1-7
P, A	DE 101 53 798 A (HENNCHEN NORBERT) 2 January 2003 (2003-01-02) column 1, line 38 - line 64 column 4, line 23 - line 54; figure 1 --- -/--	1-7

☒ Further documents are listed in the continuation of box C.

☒ Patent family members are listed in annex.

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- *L* document which may throw doubts on priority claim(s) or which is cited to establish the publication date of another citation or other special reason (as specified)
- *O* document referring to an oral disclosure, use, exhibition or other means
- *P* document published prior to the international filing date but later than the priority date claimed

- *T* later document published after the international filing date or priority date and not in conflict with the application but cited to understand the principle or theory underlying the invention
- *X* document of particular relevance; the claimed invention cannot be considered novel or cannot be considered to involve an inventive step when the document is taken alone
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- *8* document member of the same patent family

Date of the actual completion of the international search

23 May 2003

Date of mailing of the international search report

03/06/2003

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HERNANDEZ, R

INTERNATIONAL SEARCH REPORT

International Application No

PCT/EP 03/03047

C.(Continuation) DOCUMENTS CONSIDERED TO BE RELEVANT

Category *	Citation of document, with indication, where appropriate, of the relevant passages	Relevant to claim No.
A	US 5 904 228 A (STOEVER GUY T ET AL) 18 May 1999 (1999-05-18) column 4, line 10 -column 5, line 10 column 6, line 24 - line 40; figures 2,3 -----	1-7
A	US 4 254 845 A (BRAUN GUENTHER) 10 March 1981 (1981-03-10) column 5, line 17 - line 54; figure 1 -----	1-7

INTERNATIONAL SEARCH REPORT

Information on patent family members

International Application No

PCT/EP 03/03047

Patent document cited in search report		Publication date	Patent family member(s)	Publication date
US 6254197	B1	03-07-2001	AU 729248 B2	25-01-2001
			AU 5048398 A	22-06-1998
			WO 9823474 A1	04-06-1998
			EP 0939712 A1	08-09-1999
DE 10153798	A	02-01-2003	DE 10153798 A1	02-01-2003
			WO 03040556 A1	15-05-2003
US 5904228	A	18-05-1999	US 5738142 A	14-04-1998
US 4254845	A	10-03-1981	DE 2739994 B1	14-09-1978
			FR 2401808 A1	30-03-1979
			SE 412727 B	17-03-1980
			SE 7809307 A	07-03-1979

DERWENT-ACC-NO: 2003-767840

DERWENT-WEEK: 200432

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TITLE: Hydraulic brake system for wind energy plants, has drive unit to open on/off valve in discharge and supply line when rotational speed of component to be braked decreases to preset value

INVENTOR: UPHUES U

PATENT-ASSIGNEE: GENERAL ELECTRIC CO[GENE]

PRIORITY-DATA: 2002DE-1013097 (March 23, 2002)

PATENT-FAMILY:

PUB-NO	PUB-DATE	LANGUAGE
WO 03080414 A1	October 2, 2003	EN
AU 2003214254 A1	October 8, 2003	EN

DESIGNATED-STATES: AE AG AL AM AT AU AZ BA BB BG BR
BY BZ CA CH CN CO CR CU CZ DE DK
DM DZ EC EE ES FI GB GD GE GH GM
HR HU ID IL IN IS JP KE KG KP KR KZ LC
LK LR LS LT LU LV MA MD MG MK MN
MW MX MZ NI NO NZ OM PH PL PT RO
RU SC S D SE SG SK SL TJ TM TN TR
TT TZ UA UG US UZ VC VN YU ZA ZM
ZW AT BE BG CH CY CZ DE DK EA EE
ES FI FR GB GH GM GR HU IE IT KE LS
LU MC MW MZ NL OA PT RO SD SE SI
SK SL SZ TR TZ UG ZM ZW

APPLICATION-DATA:

PUB-NO	APPL-DESCRIPTOR	APPL-NO	APPL-DATE
WO2003080414A1	N/A	2003WO-EP03047	March 24, 2003
AU2003214254A1	Based on	2003AU-214254	March 24, 2003

INT-CL-CURRENT:

TYPE	IPC DATE
CIPS	B60T13/22 20060101
CIPS	F03D7/02 20060101

ABSTRACTED-PUB-NO: WO 03080414 A1**BASIC-ABSTRACT:**

NOVELTY - The system has piston/cylinder unit (14) that is supplied with a working fluid from a reservoir (22) by a supply line (18). A pump and an on/off valve are set in supply lines (18, 42) and discharge line that are linked to the piston for supplying and discharging fluid, respectively. A valve drive unit (48) closes the on/off valve when the rotational speed of a component to be braked decreases to a preset value.

USE - Used for braking moving component of wind energy plant e.g. brake disk of power train, azimuth drive of wind energy plant.

ADVANTAGE - The on/off valve arranged in the discharge line prevents the working fluid from suddenly flowing out of the cylinder/piston unit. Smooth falling of fluid into the operative position of the brake reduces vibrations, excessive dynamic load peaks, thereby protecting the system against damage and reducing material fatigue.

The working fluid in the piston/cylinder unit is kept under pressure, thus maintaining brake in the open position against the biasing action of the piston.

DESCRIPTION OF DRAWING(S) - The drawing shows a circuit diagram for a hydraulic brake system comprising a passive brake.

Brake (12)

Piston/cylinder unit (14)

Supply lines (18, 42)

On/off valves (34, 38)

Drive unit (48)

CHOSEN-DRAWING: Dwg.1/4

TITLE-TERMS: HYDRAULIC BRAKE SYSTEM WIND
ENERGY PLANT DRIVE UNIT OPEN VALVE
DISCHARGE SUPPLY LINE ROTATING
SPEED COMPONENT DECREASE PRESET
VALUE

DERWENT-CLASS: Q18 Q55 X15

EPI-CODES: X15-B09;

SECONDARY-ACC-NO:

Non-CPI Secondary Accession Numbers: 2003-615043